

PATENT

IN THE UNITED STATES PATENT AND TRADEMARK OFFICE

Application of:)	<u>CERTIFICATE OF MAILING</u>
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EDMUND W. BROWN)	being deposited with the United States
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Serial No.: 09/769,590)	envelope addressed to: Assistant
)	Commissioner for Patents and Trademarks,
Filed: January 25, 2001)	Box Amendment, Washington, D.C.
)	20231 on _____, 2002.
Examiner: Mathew C. Graham)	
)	
Art Unit: 3683)	Christine Kierzek
)	
DAMPENING CYLINDER FOR)	
TRANSFER MECHANISM)	Signature _____ Date _____

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GROUP 3600

PROPOSED RESPONSE TO OFFICE ACTION

Assistant Commissioner for Patents
Box Amendment
Washington, D.C. 20231

Sir:

Responsive to the Office Action dated July 19, 2002, applicant respectfully requests reconsideration in view of the following comments.

Claims 22-30, 32, 34, 35 and 37-39 have been rejected under 35 USC § 102(b) as being anticipated by British Publication 1,257,827. However, as hereinafter described, it is believed that such claims are not shown or suggested by the '827 reference. As such, withdrawal of the Examiner's rejection under 35 USC § 102(b) is respectfully requested.

Claim 22 defines a dampening cylinder having a cylindrical housing, a piston slidably extending through a cavity in the housing and a flange projecting from the piston so as to divide the cavity in the housing into first and second portions. A flow conduit has a first end communicating with the first portion of the cavity and a second end communicating with the second portion of the cavity. The flow conduit includes first and second control valves for controlling the flow of fluid between first and second portions of the cavity. Each flow control

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valve includes a flow regulator having a plurality of user selectable discrete settings for controlling the rate at which the fluid flows through a corresponding flow control valve. As hereinafter described, nothing in the cited references shows or suggests providing flow regulators in the flow control valves to control the rate at which fluid flows therethrough.

The British '827 specification discloses a device for balancing the forces inertia of reciprocating stands of cold rolling mills. As best seen in Figures 2 and 3, an air cylinder is provided having a piston slidably received therein which defines first and second working spaces in the air cylinder. The air spaces are interconnected by a conduit that includes first and second maximum pressure valves. It is intended that the pressure in each working space be equal. In order to accomplish the task, each of maximum pressure valves 35 is adjusted so as to satisfy condition:

$$C \cdot X = P_m \cdot S$$

wherein C and X are the rigidity and deformation of spring 50, respectively; P_m is the maximum specific air pressure in the working spaced for a given steady rolling process; S is the effective area of disk valve 48.

If movement of the piston disturbs the equality, one of the maximum pressure valves 35 opens and the excess air is transferred from one working space to the other. As described, it can be appreciated that the structure provided the British '827 specification merely regulates the maximum pressure within opposite sides of air cylinder 1. In other words, the pressure valves disclosed in the British '827 specification merely provide a mechanism for controlling the pressure at which maximum pressure vanes 35 open. The British '827 specification provides no mechanism for controlling the rate of air flow through the conduit interconnecting the first and second working spaces. Consequently, in such circumstances wherein controlled movement of the piston is required, the structure disclosed in the British '827 specification is inadequate. For example, if a large force is placed on one end of the piston that urges the piston to slide through a

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cylinder at one rate, it may be highly desirable when the force is removed to have the piston to return to its original position, at a second, slower rate. This operation can be significant in certain applications where the cylinder controls movement of an object like the transfer deck described in the specification of the present application. For example, when heavy articles are positioned on the transfer deck, it is imperative the deck travel at such a speed as to not injure the operator thereof. However, if a heavy article is placed on a deck utilizing the maximum pressure valves disclosed in the '827 reference and such article generates greater than the maximum pressure, the pressure valves would open such that air would quickly flow from one side of the cylinder to the other. As a result, the transfer deck would drop quickly in a hazardous manner.

Similarly, when the articles are subsequently removed from the transfer deck, it is imperative that the deck return to its original position at such a speed as to not injure the operator thereof. If the heavy object was removed from the deck utilizing the maximum pressure valves disclosed in the '827 specification, the deck would quickly return to its original position (in order to reach equilibrium). This, in turn, could provide a significant hazard to the operator. By providing flow regulators, as required by independent claim 22, an operator has the ability to control the speed at which the piston slides through the cylinder, thereby overcoming the disadvantages associated with the structure disclosed in the cited reference. In view of the foregoing, it is believed that independent claim 22 defines over the cited reference and passage to allowance is respectfully requested.

Claims 23-29 depend either directly or indirectly from independent claim 21 and further define a dampening cylinder not shown or suggested in the prior art. It is believed that claims 23-29 are allowable as depending from an allowable base claim and in view of the subject matter of each claim.

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Claim 30 defines a dampening cylinder incorporating a cylindrical housing and a piston slidable through the housing. A control valve structure is provided to control the flow of fluid between first and second portions of a cavity in the housing. The control valve structure includes a control valve with a flow regulator having a plurality of user selectable settings. The flow regulator controls the rate of which fluid flows through the first flow control valve. The flow control valve structure also includes a second control valve with a flow regulator having a plurality of user selectable settings. The flow regulator of the second control valve controls the rate of at which the fluid flows through the second control valve.

As heretofore described with respect to claim 22, the British '827 specification does not suggest providing control valves having flow regulators with user selectable settings which control the flow rate of the fluid through the control valves. Such a structure is entirely absent from the cited reference. As such, it is believed that independent claim 30 defines over the cited reference and is in proper form allowance.

Claims 32, 34-35 and 37 depend either directly or indirectly from independent claim 30 and further define a dampening cylinder not shown or suggested in the prior art. It is believed that such claims are allowable as depending from an allowable base claim and in view of the subject matter of each claim.

Similar to claim 30, claim 38 defines a dampening cylinder having first and second control valves. Each control valve includes a flow regulator having a plurality of user selectable settings. The settings of the flow regulators control the rates at which the fluid flows through the corresponding flow control valves. As described with respect to independent claims 22 and 30, such a structure is not shown or suggested in the cited reference. As such, it is believed that independent claim 38 defines over the cited reference and passage to allowance is respectfully requested.

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Claim 39 depends directly from independent claim 38 and further defines a dampening cylinder not shown or suggested in the prior art. It is believed that claim 39 is allowable as depending from an allowable base claim and in view of the subject matter of the claim.

Applicant believes that the present application with claims 22-30, 32, 34-35, and 37-39 is in proper form for allowance and such action is earnestly solicited.

The Applicant believes there are no fees associated with this transmission. However, the Commissioner is hereby authorized to charge payment of any fee associated with this or any other communication or credit any overpayment to Deposit Account No. 50-1170.

Respectfully submitted,

Peter C. Stomma
Registration No. 36,020

Dated: _____
BOYLE, FREDRICKSON, NEWHOLM,
STEIN & GRATZ, S.C.
250 Plaza, Suite 1030
250 East Wisconsin Avenue
Milwaukee, WI 53202
Telephone: (414) 225-9755
Facsimile: (414) 225-9753
Docket No.: 328.002



BOYLE
FREDRICKSON
NEWHOLM
STEIN & GRATZ

INTELLECTUAL PROPERTY LAW

SUITE 1030
250 EAST WISCONSIN AVENUE
MILWAUKEE, WI 53202
PHONE: 414.225.9755 FAX: 414.225.9753
E-MAIL: MAIL@BOYLEFRED.COM

FACSIMILE TRANSMITTAL SHEET

TO: EXAMINER MATHEW GRAHAM FROM: PETER C. STOMMA
COMPANY: USPTO DATE: SEPTEMBER 20, 2002
FAX NO.: 703 308 3519 PAGES: 6
RE: SN 09/769,590 OPERATOR: MCK

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Serial No.: 09/769,590
Filed: January 25, 2001
Inventor: Edmund W. Brown
Title: Dampening Cylinder For Transfer Mechanism
Art Unit: 3683

Dear Examiner Graham:

Per our telephone discussion, attached for your review is a Proposed Response to Office Action for the above-identified patent application. Please be advised that I will be in Washington, DC next Monday and Tuesday, September 23 & 24, 2002. My cell phone number is 414-350-6393. If you desire, I am available at anytime during my trip to meet with you to discuss this application.

Peter C. Stomma